

HEAT TRANSFER BY CONVECTION

CONDUCTION

Mechanism of heat transfer through a solid or fluid in the **absence** any fluid motion.

CONVECTION

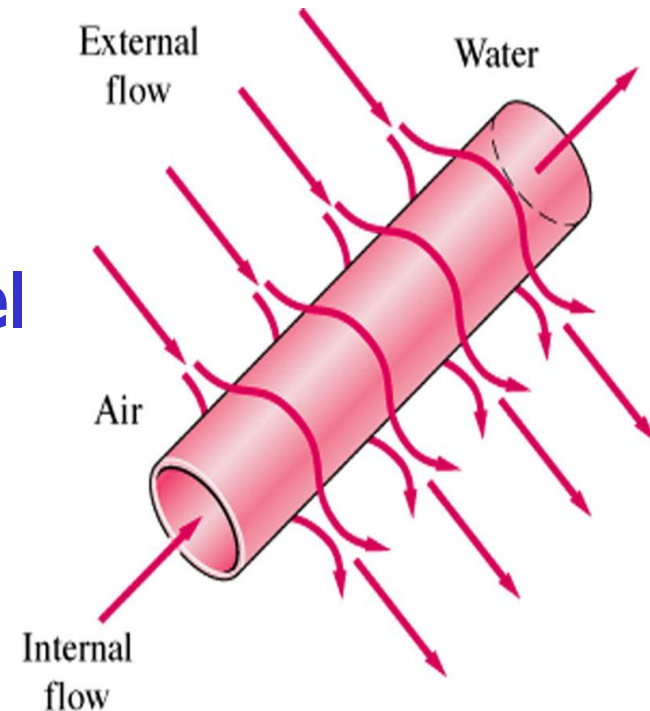
Mechanism of heat transfer through a fluid in the **presence** of bulk fluid motion

- **Natural (free) Convection**
- **Forced Convection**

(depending on how the fluid motion is initiated)

CLASSIFICATION OF FLUID FLOWS

- **Viscous-inviscid**
- **Internal flow-
External flow**
- **Open-closed channel**
- **Compressible-
Incompressible**
- **Laminar-
Turbulent**
- **Natural- Forced**
- **Steady- Unsteady**
- **One-,two-,three-
dimensional**



VISCOSITY

When two fluid layers move relative to each other, a friction force develops between them and the slower layer tries to slow down the faster layer.

internal resistance to flow

- cohesive forces between the molecules in liquid
- molecular collisions in gases.

Viscous flows: viscous effects are significant

Inviscid flow regions: viscous forces are negligibly small compared to inertial or pressure forces.

→ measure of stickiness or resistance to deformation

1. **Kinematic viscosity**
2. **Dynamic viscosity**

VISCOSITY DEPENDS ON

- **TEMPERATURE**
- **PRESSURE**

For liquids dependence of pressure is negligible

For gases kinematic viscosity depends on pressure since its relation to density

μ Dynamic viscosity
(kg/m.s or poise)

$$\nu = \frac{\mu}{\rho}$$

Air at 20°C and 1 atm:

$$\mu = 1.83 \times 10^{-5} \text{ kg/m} \cdot \text{s}$$

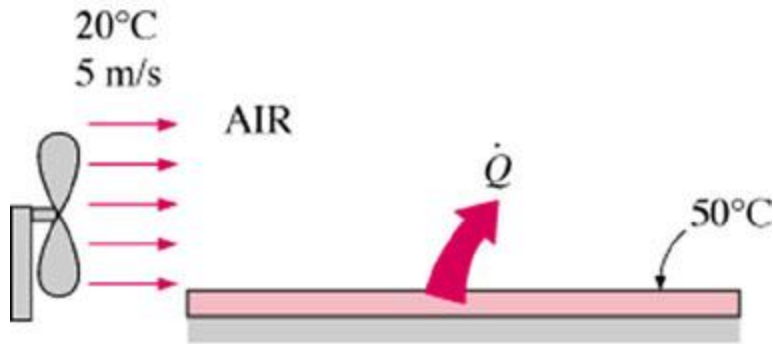
$$\nu = 1.52 \times 10^{-5} \text{ m}^2/\text{s}$$

Air at 20°C and 4 atm:

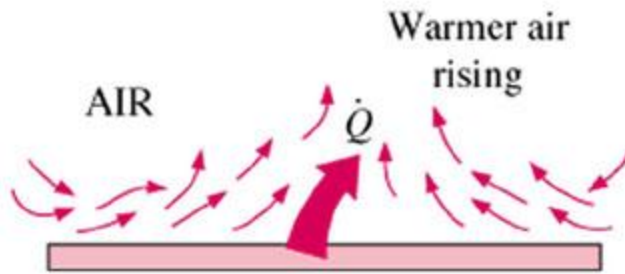
$$\mu = 1.83 \times 10^{-5} \text{ kg/m} \cdot \text{s}$$

$$\nu = 0.380 \times 10^{-5} \text{ m}^2/\text{s}$$

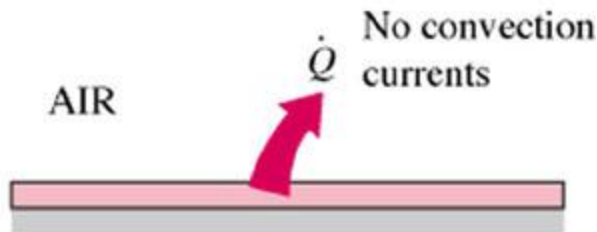
ν Kinematic viscosity,
m²/s or stroke



(a) Forced convection



(b) Free convection



(c) Conduction

Convection heat transfer

- Dynamic viscosity
- Thermal conductivity
- Density
- Specific heat
- Fluid velocity
- Geometry
- Roughness
- Type of fluid flow

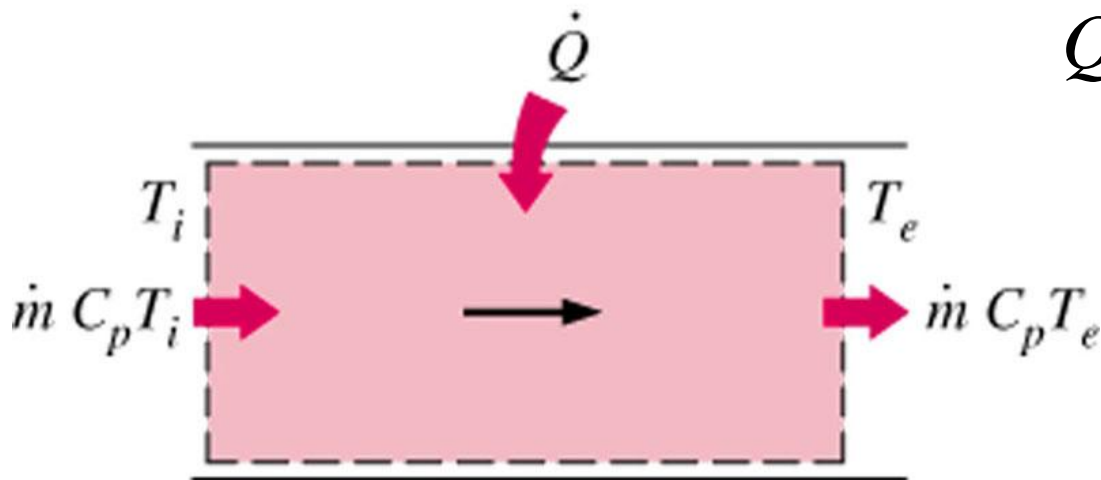
NEWTON'S LAW OF COOLING

$$\dot{Q}_{conv} = hA_S (T_S - T_\infty) \quad (\text{W})$$

h Convection heat transfer coefficient ($\text{W}/\text{m}^2 \cdot ^\circ\text{C}$)

The rate of heat transfer between a solid surface and a fluid per unit surface area per unit temperature difference

GENERAL THERMAL ANALYSIS



$$\dot{Q}_{conv} = hA_S (T_S - T_\infty)$$

$$\dot{Q} = \dot{m} C_p (T_e - T_i)$$

Energy balance:

$$\dot{Q} = \dot{m} C_p (T_e - T_i)$$

FORCED CONVECTION

- **LAMINAR FLOW**

Smooth streamlines

Highly- ordered motion

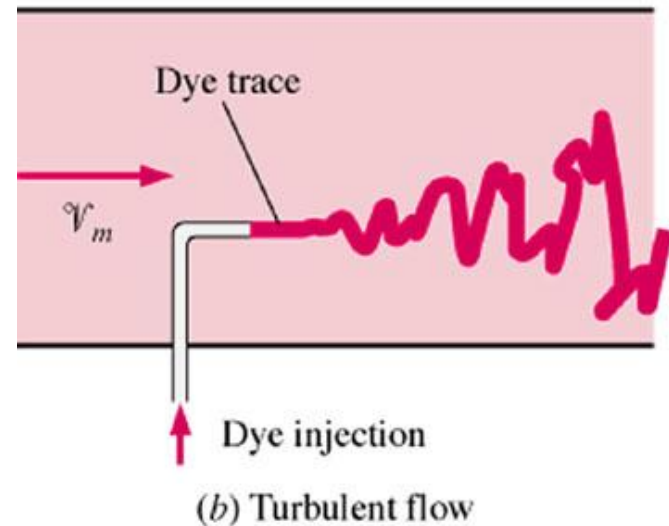
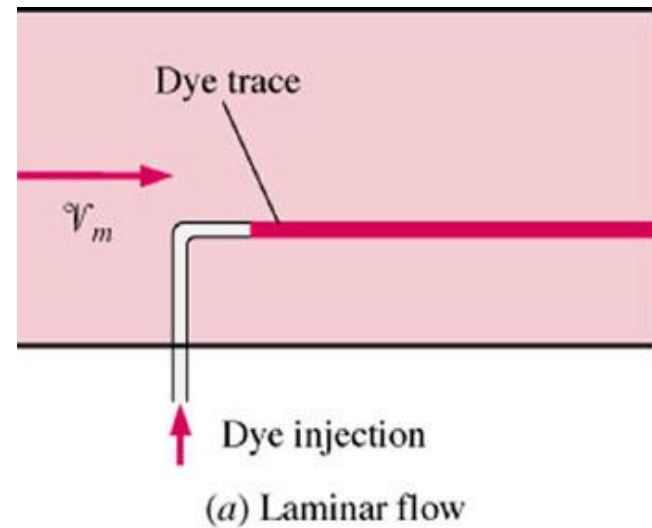
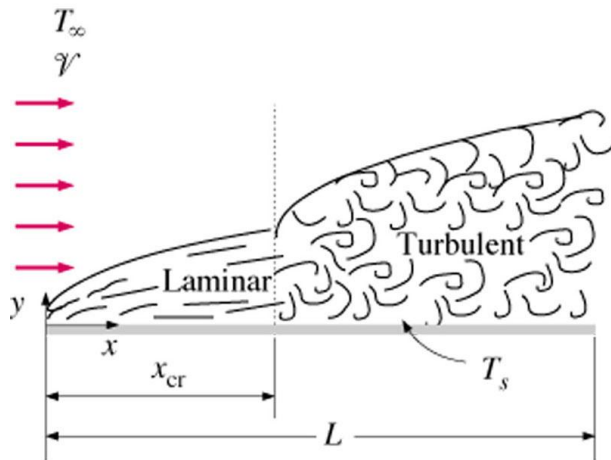
(highly viscous fluids in small pipes)

- **TURBULENT FLOW**

Velocity fluctuations

Highly-disordered motion

- **TRANSITIONAL FLOW**



REYNOLDS NUMBER

Flow Regime: Ratio of the inertial forces to
viscous forces in the fluid

Geometry

Surface roughness

Flow velocity

Surface temperature

type of fluid

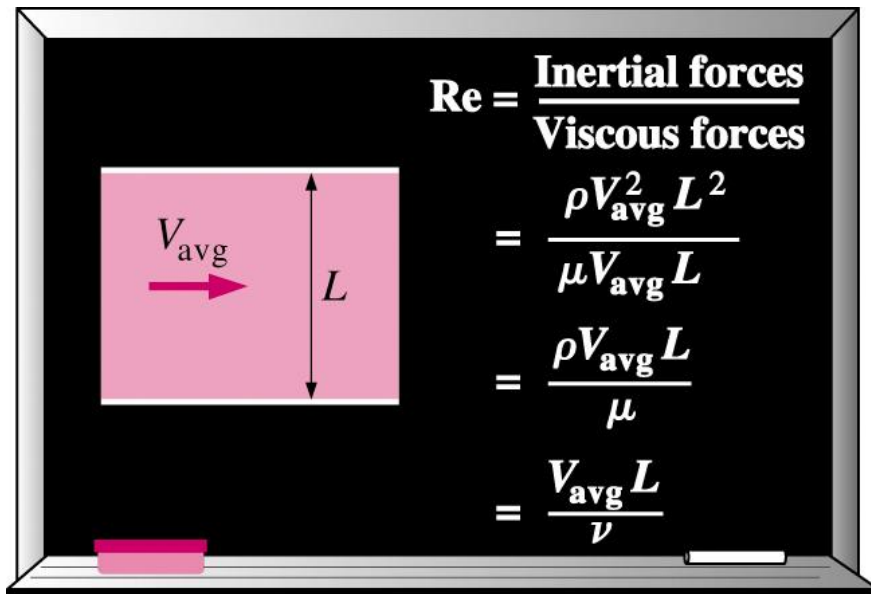
$$\text{Re} = \frac{v_m D}{\nu} = \frac{\rho v_m D}{\mu}$$

v_m Mean flow velocity

D Characteristic length of
the geometry

$\nu = \mu / \rho$ Kinematic viscosity

Definition of Reynolds number



- Critical Reynolds number (Re_{cr}) for flow in a round pipe

$\text{Re} < 2300 \Rightarrow$ laminar

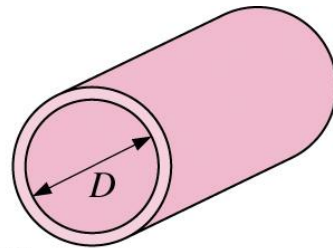
$2300 \leq \text{Re} \leq 4000 \Rightarrow$ transitional

$\text{Re} > 4000 \Rightarrow$ turbulent

- Note that these values are approximate.
- For a given application, Re_{cr} depends upon
 - Pipe roughness
 - Vibrations
 - Upstream fluctuations, disturbances (valves, elbows, etc. that may disturb the flow)

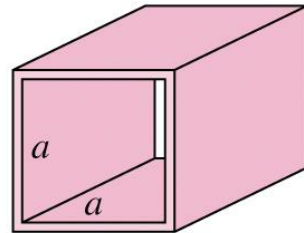
HYDRAULIC DIAMETER

Circular tube:



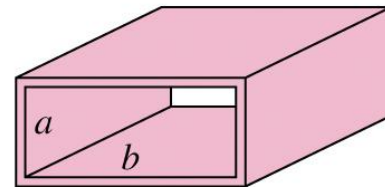
$$D_h = \frac{4(\pi D^2/4)}{\pi D} = D$$

Square duct:



$$D_h = \frac{4a^2}{4a} = a$$

Rectangular duct:



$$D_h = \frac{4ab}{2(a+b)} = \frac{2ab}{a+b}$$

- For non-round pipes,
- the hydraulic diameter

$$D_h = 4A_c/P$$

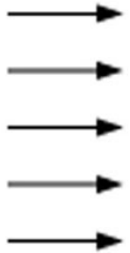
A_c = cross-section area

P = wetted perimeter

Velocity

No-slip condition

Uniform approach velocity, V



Relative velocities of fluid layers



Zero velocity at the surface

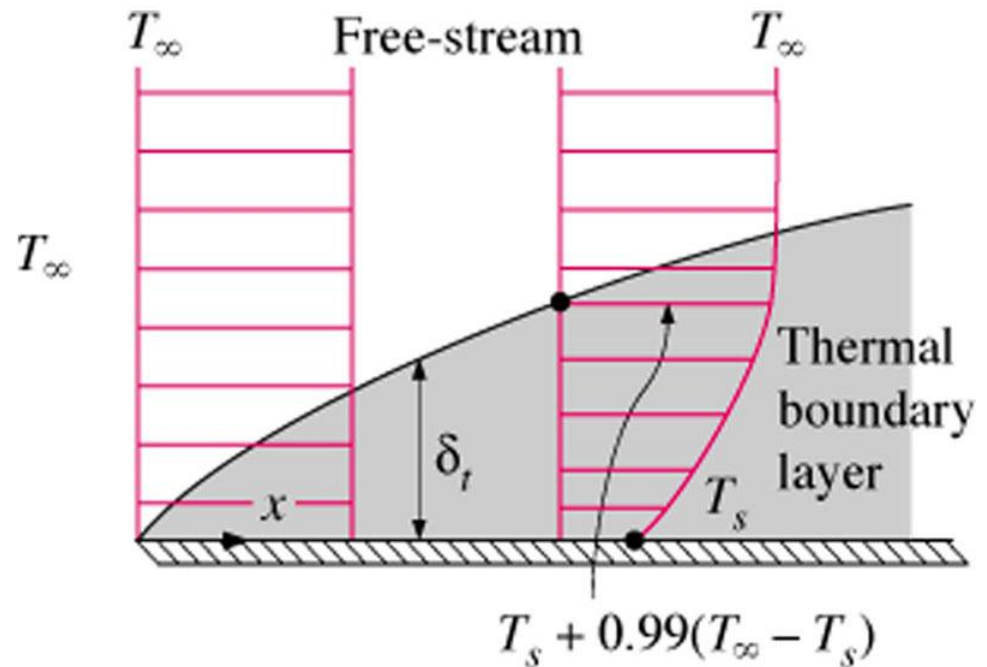


Solid block

THERMAL BOUNDARY LAYER

Flow region over the surface in which the temperature variation in the direction normal to the surface

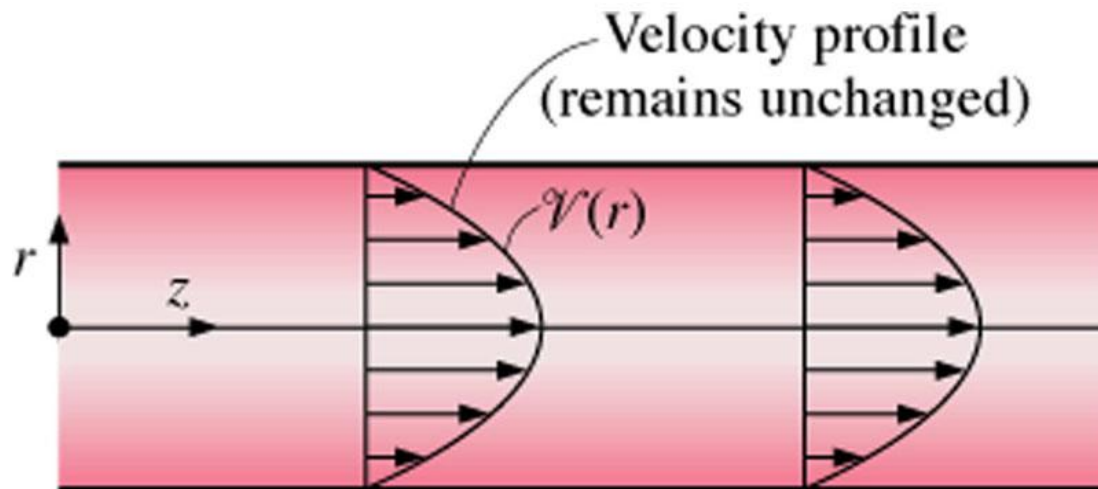
Velocity profile influences temperature profile



VELOCITY

A flow field is best characterized by the velocity distribution, and velocity may vary in three dimension

$\vec{V}(x, y, z)$ in rectangular $\vec{V}(r, \theta, z)$ in cylindrical coordinates

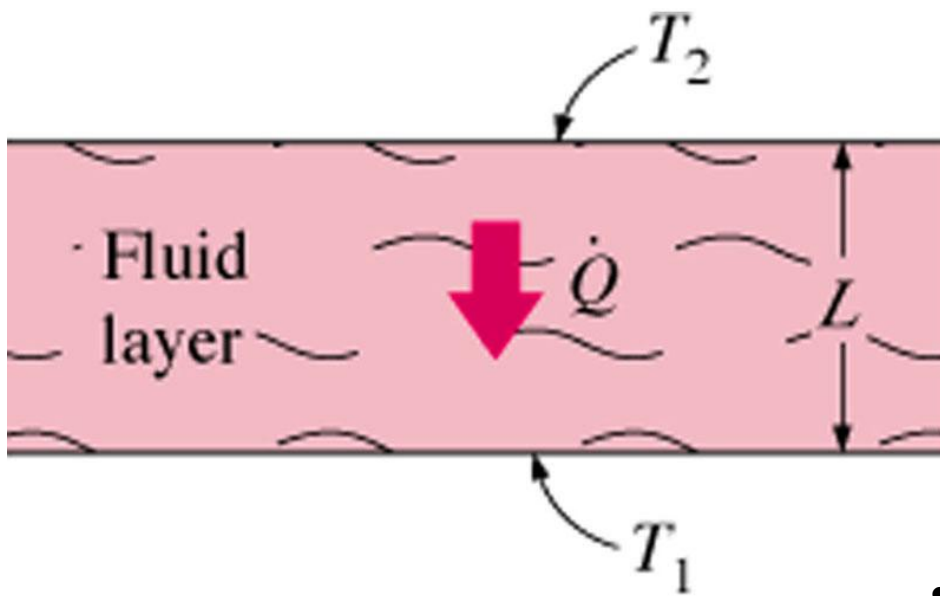


One dimensional flow in a circular pipe

In which direction does the velocity change in this figure???

NUSSELT NUMBER

(Dimensionless number)



$$\Delta T = T_2 - T_1$$

$$Nu = \frac{hL_c}{k}$$

- $q_{cond} = k \frac{\Delta T}{L}$

- $q_{conv} = h\Delta T$

- $\frac{q_{conv}}{q_{cond}} = \frac{h\Delta T}{k\Delta T / L} = \frac{hL}{k} = Nu$

PRANDTL NUMBER

- Boundary layer theory

$$\text{Pr} = \frac{\mu C_p}{k}$$

$$\text{Pr} = \frac{\text{molecular diffusivity of momentum}}{\text{molecular diffusivity of heat}} = \frac{\nu}{\alpha} = \frac{\mu C_p}{k}$$

$\text{Pr} \ll 1$ heat diffuses very quickly in liquid metals, t_b/l thicker

$\text{Pr} \gg 1$ heat diffuses very slowly in oils relative to momentum, t_b/l thinner than $\nu b/l$

PARALLEL FLOW OVER FLAT PLATES

$$\text{Re}_{cr} = \frac{\rho v x_{cr}}{\mu} = 5 \times 10^5$$

$$Nu = \frac{hL}{k} = 0.664 \text{ Re}_L^{0.5} \text{ Pr}^{1/3} \quad \text{Re}_L < 5 \times 10^5 \quad \text{laminar}$$

$$Nu = \frac{hL}{k} = 0.037 \text{ Re}_L^{0.8} \text{ Pr}^{1/3} \quad 0.6 \leq \text{Pr} \leq 60 \quad \text{turbulent}$$
$$5 \times 10^5 \leq \text{Re}_L \leq 10^7$$

NATURAL CONVECTION

CONVECTIVE HEAT TRANSFER COEFFICIENT

Coefficient of volume expansion

Grashof number

$$Gr_L = \frac{g\beta(T_s - T_\infty)L_C^3}{\nu^2}$$

viscosity

Rayleigh number

$$Ra_L = Gr_L Pr$$

Prandtl number

Nusselt number

$$Nu = \frac{hL_C}{k} = CRa_L^n$$

Table 20-1